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Building Energy Modeling: The Foundation of a GHX Design

Connor Dacquay, P.E. *GEOptimize, Inc.*

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Building Energy Modelling The Foundation of GHX Design

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Learning Objectives

• Importance of schedules inside energy models

• Difference between sum of the peak loads vs. block loads

• Difference between reduced weather data vs full weather data



Earth Energy Resource

- Constant earth temperature
- Variable outdoor air temperature





Earth Energy Resource

- Plastic pipe transfers heat to volume of earth
- Heat exchange fluid is pumped through the pipe
- Pipe is connected to a heat pump





- Pipeline large enough to meet peak heating load on the coldest day
- Cooling tower dissipates heat by evaporating water
- Energy available if utility is paid





Design and Operating Conditions		Water Distribution System Construction Materials						
Tower Type:	Counter Flow	w Induced Draft	Stand Pipe:	PVC				
Water Flow Rate (GPM):	88 GPM		Sprinkler Head:	Nylon				
Entering Water Temperature	95°F		Sprinkler Pipes:	PVC				
Leaving Water Temperature	85°F		Mechanical Equipment					
Wet Bulb Temperature:	75°F		Fan Unit:	One Unit per Tower				
Total Fan BHP:	1 HP		Туре:	Axial Flow				
Total Pump Head:	6.0'		Manufacturer:	CTS				
Drift Loss of Water Flow:	0.1%		Diameter:	30.25"				
Evaporation Loss of Water Flow:	0.93%		Blade Material:	Nylon				
Design Wind Load:	30.7 lbs/sq. f	ft.	Hub Material:	Nylon				
Structural Details			Nominal Air Volume:	8,100 CFM				
Overall Diameter:	62.25"		Fan Motor					
Overall Height:	68 3/8"	25		wer				
Dry Weight:	223 lbs.	-						
Operating Weight:	1,205 lbs.							
Basic Tower Construction Materi	als							
Tower Support Frame Assembly	Support Frame Assembly FRP		anter a state of the	and the second				
Casing:	FRP		evapco					
Casing Supporters	Nylon							
Cold Water Basin	FRP			6. A				
Filling:	PVC							
Filling Supports:	PVC							
Fan Guard	HDGS							
Mechanical Equipment Supports:	HDGS	-		5				
Inlet Louvers:	PVC	Martin Contraction		1				
Bolts, Nuts & Washers:	STS		Water Flow (GPM):	88 GPM				

- Manufacturer designs cooling towers or aircooled condensers to meet peak loads at peak design conditions
- Designers need the peak cooling load to select a cooling tower from a website or catalog

Inlet Pressure: 5 psig												
Pressure Drop: 0.5 psig												
Length	h Calculated Flow (Natural Gas)											
(ft)	(Cubic Feet of Gas Per Hour)											
	CTS IPS											
Nom OD	1/2	1	3/4	1 1-1/4		1-1/2	2	2 3				
DR	7	11.5	11	11	10	11	11	11.5	11.5			
ID	0.436	0.918	0.848	1.062	1.308	1.534	1.917	2.855	3.670			
10	481	3413	2771	5007	5007 8672 13181 23704		67535	130791				
20	331	2346	1904	3442	5960	9059 16292		46416	89892			
30	266	1884	1529	2764	4786	7275	13083	37274	72186			
40	227	1612	1309	2365	4096	6226	11197	31902	61782			
50	202	1429	1160	2096	3631	5518	9924	28274	54756			
60	183	1295	1051	1899	3290	5000	8992	25618	49613			
70	168	1191	967	1747	3026	4600	8272	23568	45643			
80	156	1108	899	1626	2815	4279	7696	21926	42462			
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200	95	675	54	54								
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300	76	542	441			-	-		20771			
350	70	499	40	a training	115				19109			
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900	42	299	24 11464									
1000	40	283	22			1 A		A.	10829			
1500	32	227	184	333	577	876	1576	4490	8696			
2000	27	194	158	285	493	750	1349	3843	7443			

- Gas utility provides tables to select gas pipe size needed for selected equipment
- Designer need peak heating load to size the pipe line



- Equipment is selected to meet the peak heating & cooling loads
- Oversizing equipment is not cost prohibitive
- The system is designed and installed





Peak Loads vs. Total Loads

- Church, retail store and apartment
- Peak cooling loads are identical 480 kBtu/hr (40 tons)
- Peak heating loads are identical 385 kBtu/hr





Peak Loads vs. Total Loads

- Different occupancy creates different annual total heating and cooling loads
- Heating / cooling ratio impacts sustainability of GHX

	Church					Retail				Apartment				
	Clg kBtu	Clg kBtu/h <mark>r</mark>	Htg kBtu	Htg kBtu/hr		Clg kBtu	Clg kBtu/hr	Htg kBtu	Htg kBtu/hr		Clg kBtu	Clg kBtu/hr	Htg kBtu	Htg kBtu/hr
Jan	3820	8	189734	385	Jan	19906	93	89734	385	Jan	5560	25	159734	385
Feb	6202	23	135120	366	Feb	28202	110	65120	346	Feb	7840	83	112120	360
Mar	12177	76	81304	312	Mar	30177	215	41304	240	Mar	14177	185	71304	305
Apr	16800	216	36614	170	Apr	40866	285	16614	110	Apr	28866	260	30614	155
May	24640	367	11152	65	May	53946	396	3152	35	May	43946	329	11152	60
Jun	46285	446	3180	5	Jun	82094	446	180	0	Jun	72094	423	8545	45
Jul	52680	480	886	0	Jul	102358	480	0	0	Jul	92358	480	7650	43
Aug	49068	465	1725	0	Aug	102393	439	125	0	Aug	78393	447	7550	45
Sep	38560	314	5479	53	Sep	89245	360	2379	26	Sep	59450	360	8479	53
Oct	13821	121	24702	137	Oct	63821	223	9702	128	Oct	19821	169	18702	132
Nov	7571	62	98784	298	Nov	41571	135	36784	251	Nov	8690	79	66784	269
Dec	4884	10	176775	348	Dec	27884	102	76775	331	Dec	6570	22	126775	340
	276508	480	765455	385		682463	480	341869	385		437765	480	629409	385
	Annual Cooling / Heating Ratio: 2.8 to 1			Annual Cooling / Heating Ratio: 2.0 to 1				Annual Cooling / Heating Ratio: 0.7 t			0.7 to 1			

Peak Loads vs. Total Loads

- GHX more similar to a battery compared to a conventional system
- GHX temperature for church and retail store fall outside efficient operating parameters
- Can't design off of peak loads and rules of thumb



Earth Energy Resource Engineering

- Conventional HVAC system does not consider total annual heating and cooling loads
- The gas line and cooling tower are sized on peak loads.





GSHP System Design Process







Building Heat Transfer – Envelope Conduction



- Heat is conducted through all surfaces and exposed to the outdoor air temperature
- The amount of heat transferred is directly proportional to the ΔT between the indoor & outdoor air temperatures



Building Heat Transfer - Internal Gains



- Occupants, lights and electrical equipment emits heat into the space
- Building use affects the amount of heat contributed by internal gains



Building Heat Transfer - Solar Gains



- Solar heat is transferred to the building through windows
- Amount of solar heat is affected by window to wall ratio and solar heat gain coefficient (SHGC) of windows
- Affected by internal and external shading devices



Building Heat Transfer - Ventilation



- Heating or cooling is required to condition fresh outdoor air
- Ventilation rate is variable based on building type
- Ventilation heating or cooling is highly dependent on climate
- HRV's affect ventilation load



Zone Loads

- Peak loads in different building zones are not coincidental
- Use of the space and exposure to environment impact peak zone load times





Block Loads

- Hourly loads for each hour are aggregated to determine peak block loads of building
- Peak block loads in this example occur at 3:00 pm





Peak Loads vs. Block Loads

- Peak loads in each zone are not coincidental and occur at different times
- Peak block loads are the loads transferred to or from the GHX
- Stacking safety factors can have unintended consequences





Energy Modelling – Schedules

- ASHRAE 90.1 for envelope U-values and internal load density
- ASHRAE 62.1 for ventilation
- Scheduling internal heat gains is sometimes challenging



Energy Modelling – Schedules – People Example



Energy Modelling – Schedules – People Example



Energy Modelling – Schedules – Lights Example



Energy Modelling – Schedules – Lights Example



Energy Modelling – Schedules – Misc Example



Energy Modelling – Schedules – Misc Example



Energy Modelling – Schedules – DHW Example



Energy Modelling – Schedules – Ventilation Example

Vent - Apartment Complex Day type: Weekday Month: January



Energy Modelling – Schedules – Ventilation Example



Energy Modelling – Schedules – Night-time Setback

- Small difference in energy savings for GSHP system
- Increase peak heating loads
- Increase in GHX size







Energy Modelling – Mental Model




Energy Modelling as a Design Tool



Case Study 1 – Office Expansion

- Increased floor area by 28%
- Increased window to wall ratio to 30%
- Existing GSHP system





Case Study 1 - Project Description





Case Study 1 - GHX Description



Case Study 1 - Annual Energy Load Variability

SHIGCC 0 250





Case Study 1 - Long-Term GSHP Temperature



Case Study 2 – Warehouse Retrofit

- 250,000 ft² manufacturing facility
- Existing natural gas heating system
- Interested in GSHP system





Case Study 2 – Energy Model V1

- Trace Trace 700 used to model building
- Floor plans and interviews with the building owners used to develop energy model





Case Study 2 – GHX Size V1

- 162 boreholes to a depth of 400 ft with 30 ft spacing
- Maintains a minimum EWT of 32°F and maximum of 90°F over 20 years



Case Study 2 – Energy Model V2

- Sensible wheel energy recovery ventilation added
- Reduces total amount of heating needed and increases the energy balance



Case Study 2 – GHX Size V2

- 90 boreholes to a depth of 400 ft with 30 ft spacing
- Maintains a minimum EWT of 32°F and maximum of 90°F over 20 years



Case Study 2 – GHX Reduction



Case Study 2 – Economics

- Vertical drilling in region was \$27/ft supplied and installed
- V2 reduces total capital cost and increase long term sustainability of GHX



Energy Modelling – Mental Model





Weather Data

- Some modelling software defaults to reduced year weather data
- Reduced weather data only includes weekday, Monday, Saturday, and Sunday
- Important to use full-year weather data (TMY, CWEC, EPW etc.)





Learning Objectives

- Importance of schedules inside energy models
 Schedules allow for modelling hourly block loads of the building which creates a more accurate energy model.
- Difference between peak loads vs. block loads

Hourly loads for each hour are aggregated to determine peak block loads of building. Peak loads in each zone occur at different times but are summated.

Difference between reduced weather data vs full weather data

Reduced weather data only includes weekday, Monday, Saturday, and Sunday



Energy Modelling Software

- How many people energy model?
- What software do you use?
 - IES
 - Trane Trace
 - eQuest
 - Carrier Hap



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	2	Enter Project Information	Demo - Alt 1	Demo-Alt 2	Demo-Alt 3	Demo - Alt 4
	7	Select Weather Information	8760 Stroudsberg	8760 Stroudsberg	8760 Stroudsberg	8760 Stroudsberg
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	盘	Create Rooms	36 Rooms	Use Alternative 1	Use Alternative 1	36 Rooms
	σĽ	Create Systems	2 Systems	Use Alternative 1	Use Alternative 1	2 Systems
		Assign Rooms to Systems	36 Assigned Rooms	36 Assigned Rooms	36 Assigned Rooms	36 Assigned Rooms
		Create Plants	4 Plants	4 Plants Based on Alternative 1	4 Plants Based on Alternative 2	2 Plants
	3	Assign Systems to Plants	System Assignments	System Assignments	System Assignments	System Assignments
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Category Water source	heat pump 💌			
Equipment type WaterFurnace	• NXW Heat Pump 👤	Thermal Storage		New Equip
Sequencing type Single	•	Type GLHE design	ed for 3F (2C) TD wellfield 💌	Copy Equip
Backup heat source GAS RTU	•	Capacity 479.6	gal/ton 💌	
Reject condenser heat Ground loop		Schedule Heatpump		
Reject heat to plant	~		Curturb	
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Cooling	tons	12.9	EER (compressor only)	Breakout
Heat recovery	10.88 Mbh/ton	2.7	COP (compressor only)	Diedkout
Tank charging	tons		kW/ton	
Tank charging & heat recovery	tons		kW/ton	
Pumps	Туре	Full	load consumption	
Primary chilled water	Cnst vol chill water pump	0	ft water	
Condenser water	None	0	ft water	
Heat recovery or aux condenser	None	0	ft water	
<u>C</u> onfiguration	Cooling Equipment	Heating Equipm	nent <u>B</u> ase Utility /	/ Misc. Accessory

Cooling Equipment Contro	bls			\times				
Plant description: GSHP				ок				
Equipment tag: Air-coo	led chiller - 002		C	ancel				
Free Cooling								
Туре	None	Cooli	Looling Plant and					
Fluid cooler type	None	Geo Co	Geothermal Controls					
Pump	None	Ψ.						
Pump full load energy	0 kw							
Refrig economizer unloading curve	None	None						
Chilled None	dOn	Reset Curve	Max Reset TD	۴F				
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Load shedding economizer	No 💌	Dsn chilled water delta T	6.66667	°F				
Evaporative precooling	No 💌	Dsn cond water delta T	5.55556	*F				
Equipment schedule	Available (100%)	-						
Demand limiting priority								
Hot gas reheat for dehumidification	No	•						

Plant Controls					×
Description	GSHP				
Sizing method	Peak based on design simulation				ОК
Heat rejection					Cancel
. Туре	None			-	
. Hourly ambier	it wet bulb offset	°F Geo	thermal Loop —		
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			^P ump F.L rate	50 ft water	
Thermal storage		1	Flow scheme	Fully mixed	_
Туре	None			Number of simulation years 1	
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Fuel Rates	Condenser Water Flow Rate:1500.0 gpmCondenser Pump Head:60.0 ft wgCondenser Pump Mech. Efficiency:80.0 %Condenser Pump Elec. Efficiency:94.0 %	Jul Aug Sep Oct Nov Dec	94.8 86.1 777.5 69.1 60.0 52.5 <u>С</u> ancel <u>H</u> elp
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10 Janua	ry 1	L 6	2.3	11		6.3	-6300	6.3	0.1		2.6	2600	2.6	0.2	0	0		(
11 Janua	ry 1	L 7	3.8	2.5		6.3	-6300	6.3	0.1		2.6	2600	2.6	0.2	0	0		(
12 Janua	ry 1	L 8	4.8	3.5		9.6	-9600	9.6	.2		2.6	2600	2.6	0.2	0	0		9
13 Janua	ry 1	L 9	5.5	43		9.6	-9600	9.6	.2		2.5	2500	2.5	0.2	0	0		9
14 Janua	ry 1	L 10	6.9	5.6		9.8	-9800	9.8	.2		2.5	2500	2.5	0.2	0	0		5
15 Janua	ry 1	1 11	8.0	- 12		9.9	-9900	9.9	.2		2.4	2400	2.4	0.2	0	0		
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→ HVAC Loads, Sizing & Reports	HVAC System loads and sizing	
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	Summary	
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	Compliance route: None (Design)	
	System Summary	
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	HVAC Zones assigned: 15	
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Step 4 of 4	Export equipment schedule	
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🕤 Dane County Regional (ASHRAE Climate Zone: 5A) -473'- 5", 214'- 7" 📔 Tasks 🏢 No Notes 📟 1 Alert



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Final Remarks

- Human inconsistency avoidance tendency
- Confirmation bias
- Positive vs. zero sum mindset







NY-GEO Conference 2023-04-26

Building Energy Modelling The Foundation of GHX Design

Dr. Connor Dacquay, PE, CGD

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